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弹性半空间地基上正交异性矩形中厚板的弯曲

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摘 要:为了分析弹性半空间地基上正交异性矩形中厚板的弯曲解析解,将 3 个广义位移变量描述的弹性半空间地基上四边自由正交各向异性矩形中厚板的弯曲控制方程,与基于弹性半空间地基受任意竖向荷载作用下的静力位移积分解建立的板与地基变形协调方程相结合,用三角级数法,得出弹性半空间地基上四边自由正交异性矩形中厚板受任意竖向荷载作用下的弯曲解析解,即得出了地基反力、板的挠度及板的内力的解析表达式。研究表明,该方法克服了数值法的弊端,取消了 Winkler 地基模型或双参数地基模型的假设,从而得到板的内力及地基反力更合理、更精确的分布规律。

关键词:结构工程;弹性半空间地基;正交异性矩形中厚板;弯曲;解析解

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Bending of orthotropic rectangular middle thick plate on elastic half-space foundation

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Abstract: The bending equation of orthotropic rectangular middle thick plate with four free edges was combined with the equation of deformation compliance of the plate and the foundation based on the integral solution of the elastic half-space foundation loaded with arbitrary vertical lead force. An analytical solution was gained to the problem of the bending of the orthotropic rectangular middle thick plate with four free edges on the semi-infinite elastic foundation by using the trigonometric series, the analytical representations of the reactive force of the ground, the deflection and the inner force of the plate were obtained. The results show that the method not only overcomes some defects involved in traditional numerical methods, but also avoids the assumption of Winkler foundation model or two-parameter foundation model. The distributions law of the inner forces in the plate and the contact pressure are more reasonable and accurate. 1 tab, 4 figs, 11 refs.

Key words: structural engineering; elastic half-space foundation; orthotropic rectangular middle thick plate; bending; analytic solution

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0 引言

弹性半空间地基上四边自由矩形中厚板的弯曲问题,一直是学者研究的热点和难点问题。国内外众多学者相继采用各种方法,如有限元法、有限元-边界元混合法、无单元法和 Ritz 法,对弹性半空间地基上四边自由矩形板的弯曲特性进行了研究^[1-10]。但是,针对弹性半空间地基上正交异性矩形中厚板的弯曲解析解,却未见报道。为此,本文将 3 个广义位移描述的弹性半空间地基上四边自由正交各向异性矩形中厚板的弯曲控制方程,与基于弹性半空间地基受任意竖向荷载作用下的静力位移积分变换解建立的板和地基的变形协调方程相结合,用三角级数法,得出弹性半空间地基上四边自由正交异性矩形中厚板受任意竖向荷载作用的弯曲解析解。

1 控制微分方程及其边界条件

弹性半空间地基上长为 a 、宽为 b 的正交各向异性矩形中厚板,受垂直于板面横向分布荷载 $q(x, y)$ 作用,若地基反力为 $F(x, y)$,取 x, y 坐标轴同板的主方向平行,根据文献[11],可得控制微分方程为

$$D_{11} \frac{\partial^2 \Phi_x}{\partial x^2} + D_{66} \frac{\partial^2 \Phi_x}{\partial y^2} + (D_{12} + D_{66}) \frac{\partial^2 \Phi_y}{\partial x \partial y} + C_{11} \frac{\partial \omega}{\partial x} - C_{11} \Phi_x = 0 \quad (1)$$

$$D_{22} \frac{\partial^2 \Phi_y}{\partial y^2} + D_{66} \frac{\partial^2 \Phi_y}{\partial x^2} + (D_{12} + D_{66}) \frac{\partial^2 \Phi_x}{\partial x \partial y} + C_{22} \frac{\partial \omega}{\partial y} - C_{22} \Phi_y = 0 \quad (2)$$

$$C_{11} \frac{\partial^2 \omega}{\partial x^2} + C_{22} \frac{\partial^2 \omega}{\partial y^2} - C_{11} \frac{\partial \Phi_x}{\partial x} - C_{22} \frac{\partial \Phi_y}{\partial y} + q - F = 0 \quad (3)$$

式中: Φ_x, Φ_y 分别为变形前垂直中面的直线段在 xz 平面内和 yz 平面内的转角; ω 为挠度; D_{ij} 为弯曲刚度矩阵; C_{ij} 为剪切刚度矩阵; E_1, E_2 为弹性模量; G_{12}, G_{13}, G_{23} 为剪切模量; ν_1, ν_2 为泊松比; h 为板厚。

有如下关系式

$$D_{11} = \frac{E_1 h^3}{12(1 - \nu_1 \nu_2)}, D_{22} = \frac{E_2 h^3}{12(1 - \nu_1 \nu_2)},$$

$$D_{12} = \frac{\nu_2 E_1 h^3}{12(1 - \nu_1 \nu_2)}, D_{66} = G_{12} h^3 / 12,$$

$$C_{11} = hG_{23}, C_{22} = hG_{13}$$

板的弯矩(M_x, M_y)、扭矩(M_{xy})、剪力(Q_x, Q_y)可表示为

$$M_x = -D_{11} \frac{\partial \Phi_x}{\partial x} - D_{12} \frac{\partial \Phi_y}{\partial y},$$

$$M_y = -D_{12} \frac{\partial \Phi_x}{\partial x} - D_{22} \frac{\partial \Phi_y}{\partial y},$$

$$M_{xy} = -D_{66} \left(\frac{\partial \Phi_x}{\partial y} + \frac{\partial \Phi_y}{\partial x} \right),$$

$$Q_x = C_{11} \left(\frac{\partial \omega}{\partial x} - \Phi_x \right), Q_y = C_{22} \left(\frac{\partial \omega}{\partial y} - \Phi_y \right)$$

四边自由的矩形中厚板边界条件为

当 $x = 0$ 或 a 时, $M_x = 0, M_{xy} = 0, Q_x = 0$

当 $y = 0$ 或 b 时, $M_y = 0, M_{xy} = 0, Q_y = 0$

2 问题解析解的推求

采用文献[11]中的方法,其解统一取为

$$\omega = \sum_m \sum_n \omega_{mn} \cos(\alpha_m x) \cos(\beta_n y)$$

$$\Phi_x = \sum_m \sum_n \varphi_{mn} \sin(\alpha_m x) \cos(\beta_n y)$$

$$\Phi_y = \sum_m \sum_n \psi_{mn} \cos(\alpha_m x) \sin(\beta_n y)$$

$$q = \sum_m \sum_n q_{mn} \cos(\alpha_m x) \cos(\beta_n y)$$

$$F = \sum_m \sum_n Q_{mn} \cos(\alpha_m x) \cos(\beta_n y)$$

$$\alpha_m = m\pi/a, \beta_n = n\pi/b$$

其中, $\omega_{mn}, \varphi_{mn}, \psi_{mn}, q_{mn}, Q_{mn}$ 均为相应量的傅立叶系数, q_{mn} 已知,其余均未知。

双轴对称解取 $m, n = 0, 2, 4, \dots$; 双轴反对称解取 $m, n = 1, 3, 5, \dots$; 对称反对称解取 $m = 0, 2, 4, \dots, n = 1, 3, 5, \dots$; 反对称对称解取 $m = 1, 3, 5, \dots, n = 0, 2, 4, \dots$ 。

为满足边界条件 $Q_x|_{x=0} = 0, Q_y|_{y=0} = 0, M_{xy}|_{x=0} = 0, M_{xy}|_{y=0} = 0$, 又可在边界上连续微分,如文献[11]一样,可令

$$\frac{\partial \omega(0, y)}{\partial x} = -\frac{a}{4} \sum_n a_n \cos(\beta_n y)$$

$$\frac{\partial \omega(x, 0)}{\partial y} = -\frac{b}{4} \sum_m b_m \cos(\alpha_m x)$$

$$\Phi_x(0, y) = -\frac{a}{4} \sum_n a_n \cos(\beta_n y)$$

$$\Phi_y(x, 0) = -\frac{b}{4} \sum_m b_m \cos(\alpha_m x)$$

$$\frac{\partial \Phi_x(x, 0)}{\partial y} = -\frac{b}{4} \sum_m b_m \alpha_m \sin(\alpha_m x)$$

$$\frac{\partial \Phi_y(0, y)}{\partial x} = -\frac{a}{4} \sum_n a_n \beta_n \sin(\beta_n y)$$

其中, a_n, b_m 为待定系数。

利用傅立叶级数理论,可得 Φ_x, Φ_y, ω 的各阶偏导数表达式,并代入弯矩(M_x, M_y)、扭矩(M_{xy})及剪力(Q_x, Q_y)表达式,并考虑剩下的 2 个边界条件 $M_x|_{x=0} = 0, M_y|_{y=0} = 0$, 可得

$$\sum_m (D_{11}\epsilon_m a_n + D_{12}\epsilon_n b_m + D_{11}\alpha_m \varphi_{nm} + D_{12}\beta_n \psi_{nm}) = 0 \tag{4}$$

$$\sum_n (D_{12}\epsilon_m a_n + D_{22}\epsilon_n b_m + D_{12}\alpha_m \varphi_{nm} + D_{22}\beta_n \psi_{nm}) = 0 \tag{5}$$

将 Φ_x 、 Φ_y 、 ω 、 q 、 F 的表达式及各阶偏导数代入微分控制方程式(1) ~ 式(3),可得

$$D_{11}\epsilon_m \alpha_n a_n + (D_{12}\epsilon_n \alpha_m + D_{66}\epsilon_n \alpha_m - D_{66}\epsilon_n) b_m + (D_{11}\alpha_m^2 + D_{66}\beta_n^2 - C_{11}) \varphi_{nm} + (D_{12} + D_{66}) \cdot \alpha_m \beta_n \psi_{nm} + C_{11} \alpha_m \omega_{nm} = 0 \tag{6}$$

$$D_{12}\epsilon_m \beta_n + (D_{66}\epsilon_m \beta_n - D_{66}\epsilon_m) a_n + D_{22}\epsilon_n \beta_n b_m + (D_{12} + D_{66}) \alpha_m \beta_n \varphi_{nm} + (D_{22}\beta_n^2 + D_{66}\alpha_m^2 - \eta_{pqmn}) = \int_{-\infty}^{\infty} \int_{-\infty}^{\infty} \frac{[(-1)^p e^{i\xi a} - 1][(-1)^q e^{i\eta b} - 1][(-1)^m e^{-i\xi a} - 1][(-1)^n e^{-i\eta b} - 1]}{q_1 \xi^2 \eta^2 \left[1 - \left(\frac{m\pi}{a\xi}\right)^2\right] \left[1 - \left(\frac{n\pi}{b\eta}\right)^2\right] \left[1 - \left(\frac{p\pi}{a\xi}\right)^2\right] \left[1 - \left(\frac{q\pi}{b\eta}\right)^2\right]} d\xi d\eta$$

由矩形板的挠度与弹性地基表面竖向位移相等,可得变形协调方程为

$$\omega_{nm} - \frac{1}{2\pi^2 ab} \frac{(\lambda + 2\mu)}{(\lambda + \mu)\mu} \sum_{p=0}^{\infty} \sum_{q=0}^{\infty} Q_{pq} \eta_{pqmn} \lambda_{nm} = 0 \tag{9}$$

由式(4) ~ 式(9)这6组方程,可联立求解各待定系数 a_n 、 b_m 、 φ_{nm} 、 ψ_{nm} 、 ω_{nm} 、 Q_{nm} 。各系数求得后,不难得出板的挠度、地基反力和板的内力。

3 数值算例

算例 1:考虑一支承在弹性半空间上、边长 $a = 4\text{ m}$ 、厚度 $h = 0.2\text{ m}$ 的弹性方薄板的弯曲;假设板与地基之间为光滑接触;地基泊松比为 0.4,弹性模量为 343 MPa;板的泊松比为 0.167,弹性模量为 34 300 MPa;板上均布荷载 $q = 0.98 \times 10^6\text{ Pa}$;采用本文方法(计算 m 、 n 取到 20),计算结果见表 1。

表 1 板中心挠度和弯矩值

本文结果		文献[9]结果		样条有限元 ^[3]	
ω_{\max}/m	M_x/kN	ω_{\max}/m	M_x/kN	ω_{\max}/m	M_x/kN
0.010 7	33.168	0.010 7	35.558	0.010 62	35.512

图 1、图 2 分别是本文方法所得的挠度分布和地基反力与荷载比分布,与文献[3,9]的计算结果吻合良好。

算例 2:考虑一支承在弹性半空间地基上、边长 $a = 4\text{ m}$ 、厚度 $h = 0.2\text{ m}$ 的四边自由正交各向异性弹性方薄板的弯曲;假设板与地基之间为光滑接触;地基泊松比为 0.4,弹性模量为 $E_s = 343\text{ MPa}$;板的 x 、 y 轴泊松比分别为 $\nu_x = 0.3$ 、 $\nu_y = 0.1$,拉压弹性模量 $E_x = 34\,300\text{ MPa}$,剪切弹性模量均取为 $2.4D_{11}/h^3$, $D_{xy} = 0.2D_x$;板上中心区域(边长为 $0.1a$ 的方形区域)作用合力为 P 的均布荷载,其中

$$C_{22})\psi_{nm} + C_{22}\beta_n\omega_{nm} = 0 \tag{7}$$

$$C_{11}\alpha_m\varphi_{nm} + C_{22}\beta_n\psi_{nm} + (C_{11}\alpha_m^2 + C_{22}\beta_n^2)\omega_{nm} + Q_{nm} = q_{nm} \tag{8}$$

据文献[9],弹性半空间地基表面位移 $\omega|_{z=0}$ 可展成双重余弦级数,计算式为

$$\omega|_{z=0} = \sum_{m=0}^{\infty} \sum_{n=0}^{\infty} \lambda_{mn} \omega_{mn} \cos\left(\frac{m\pi x}{a}\right) \cos\left(\frac{n\pi y}{b}\right)$$

其中, λ_{mn} 见文献[9]。

$$\omega_{mn} = \frac{1}{2\pi^2 ab} \frac{(\lambda + 2\mu)}{(\lambda + \mu)\mu} \sum_{p=0}^{\infty} \sum_{q=0}^{\infty} Q_{pq} \eta_{pqmn}$$

其中, λ 、 μ 为地基常数,见文献[9]。

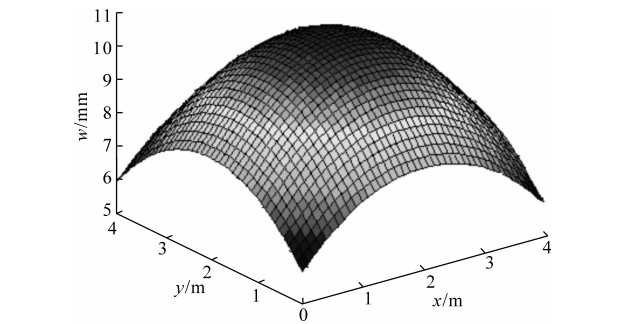


图 1 地基板的挠度 w 的分布

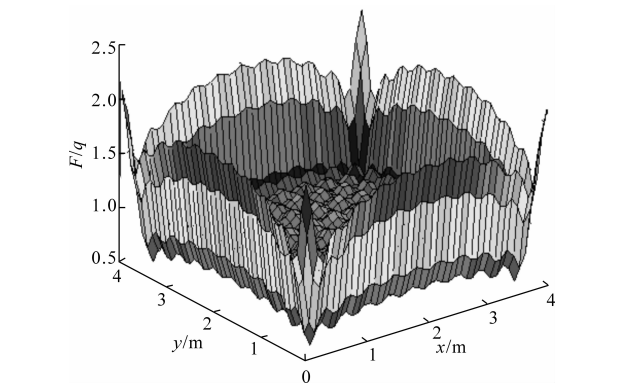


图 2 地基反力与荷载比的分布

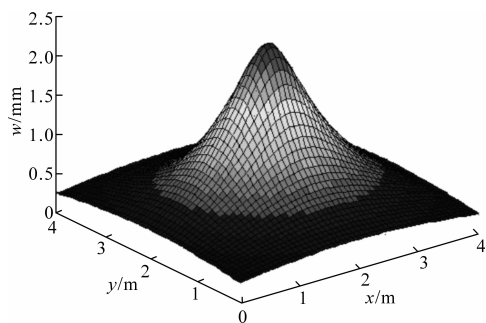
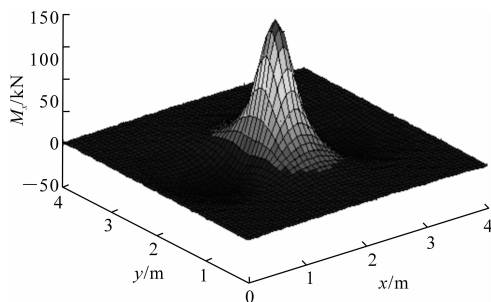
$P = 1\,000\text{ kN}$ 。

计算 m 、 n ,也取到 20,剪切弹性模量取相同值,所得结果如下页图 3 和图 4 所示。该结果与文献[10]结果吻合。

通过以上计算分析,说明本文理论推导是正确无误的,分析方法也是有效的。

4 结 语

(1)应用三角级数法,得出弹性半空间地基上四

图3 地基板的挠度 w 图4 弯矩 M_x

边自由正交异性矩形中厚板受任意竖向荷载作用的弯曲解析解,包括板的挠度、内力及板与地基之间的接触反力;这不仅克服了数值法的一些弊端,同时取消了 Winkler 地基模型或双参数地基模型的假设,从而得到板的内力及板与地基之间接触反力更合理、更精确的分布规律。

(2) 由于 η_{pqmn} 除了与 p, q, m, n 有关外,还与板的几何尺寸 a, b 有关,而与地基土及板的物理参数无关;因此,对一定尺寸(a, b)的板(其实,是对一定的长、宽比),求出的 η_{pqmn} 能用于一切弹性半空间地基上的等长宽比任意(边界条件任意、薄厚任意、性质任意)矩形板的弯曲分析;这样,就可使得弹性半空间地基上矩形中厚板这一复杂的接触问题的求解统一化、简单化和规律化。

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